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SteBLife, a New Approach for the Accelerated Generation of Metallic Materials' Fatigue Data

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Received: 27 May 2020; Accepted: 10 June 2020; Published: 16 June 2020



Abstract: The service life of materials and components exposed to repeated mechanical loads is limited, which is why the understanding of the damage evolution and estimating its fatigue life is of high importance for its technical application. This paper shows how temperature and magnetic field measurement methods can be used to describe the cyclic deformation behaviour of metallic materials and to derive parameters from this, which are used in short-term methods to calculate the fatigue life. Within the SteBLife (stepped-bar fatigue life) approach, only three to five fatigue tests with a stepped fatigue specimen are required to determine a complete S–N or Woehler curve with scatter bands for different failure probabilities. If only a trend S–N curve is required, the number of tests can be reduced to a single fatigue test only. In the framework of this paper, these approaches will be presented for normalised SAE 1045 (C45E) and quenched and tempered SAE 4140 (42CrMo4) steels.

Keywords: fatigue life evaluation; SteBLife; thermography; magnetic-based measurements

1. Introduction

In most cases, component failure occurs due to a combination of static, quasistatic, dynamic and impact loads (forces, moments, temperature, etc.), which cause increasing stresses as well as strains, resulting in micro-, and in the following, macrocracks and, finally, in failure. Since fatigue can be a subject of large scatter, engineering is striving for enhancement in the precision of fatigue life estimation to the utmost possible [1–4]. The high effort for generating fatigue data of real engineering structures has led to a large number of simplifications in the past, such as using tensile strength, hardness or impact energy to characterise fatigue in lack of any better means. These approaches are still applied in some specific fields even today, leading to high safety factors and consequently high component weights and costs. In order to be able to reduce these safety factors, more appropriate means to generate fatigue data are required, such as considering fatigue-induced microstructural changes into the component design.

Only since the mid-1990s, the use of fatigue data for the design of cyclic or service loaded components became popular [1,5,6]. The specimen or component lifetime depends on fatigue-life-influencing parameters, such as load, material, manufacturing, structural geometry as well as ambient conditions, acting individually or in a combination. Therefore, it is of prime importance



to determine these data quickly and cost-effective with respect to a component's individual loading condition. Pawliczek and Rozumek, presented in [7] an algorithm for calculating the fatigue life of specimens taking into account the effect of the mean load, their model is based on a cycle asymmetry sensitivity factor, and can be used for tension and for torsion. The proposed model allows determining the equivalent stress amplitude in a symmetrical cycle (R = -1) for the given amplitude values and the average stress value of the asymmetrical cycle (R, -1).

Damage normally occurs in the areas of highest stress concentrations, so-called hotspots, so these areas are of main interest for the design process as well as for the selection of structural material. Structural durability related design requires defined criteria such as a service life goal, reliability and functional capability as well as operational safety for the use until reaching the specified lifetime for a required calculated probability of failure [1,2,5]. The fatigue behaviour is described in the case of constant amplitude loading by S–N curves and under variable amplitude loading by the fatigue life or Gassner curves. The position of a Gassner curve relative to the S–N curve depends on the load spectrum applied [1,8].

The fatigue life of a service loaded component can be predicted from the classical point of view by using a Palmgren-Miner damage accumulation rule [9,10]. Here, each of the cycles is assigned to partial damage summed up to the total damage amount indicating a possible failure when reaching a certain damage threshold value. Depending on how the damage parameter considered for accumulation develops over the fatigue life, it is classified as linear or not. However, this hypothesis, as well as various of its modifications which have gained practical relevance, still depend strongly on loads and their related levels, which cannot be found to be the only representative parameter with regard to damage [11–13].

A first step in enhancing such an understanding was achieved with the introduction of the so-called local strain approach [14,15]. This approach is based on the idea that a structural component's material behaviour in a notch can be transferred onto a simple unnotched specimen of the same material. The main advantage of this approach is that only the material's fatigue life curve has to be determined experimentally while the S–N curves of the different structural components can be determined on an analytic and numeric basis [14]. A further advantage of the local strain approach is that the stress-strain behaviour in a notch root can be expressed and hence understood from the stress-strain hysteresis loops generated through the transformation process using rules established by Neuber [11] and others [14,15]. This at least allows the influence of plastic strains occurring in a notch root to be understood and better appreciated, which are considered to be a much more relevant fatigue accumulation parameter than stresses. Plotting the evolution of the plastic strain amplitude as a function of the number of load cycles applied to a material or a component as a first indicator of a parameter accumulating on a nonlinear basis is, therefore, made available [16].

S–N curves expressed as a function of the stress amplitude versus the number of cycles to failure in a logarithmic or double logarithmic scale are a fact in structural design that is hard to remove nowadays. The S–N curve for stress-controlled fatigue tests is usually described through the Basquin [17] relationship. For strain-controlled fatigue tests, the Manson-Coffin [18] relationship can be used to describe the low cycle fatigue (LCF) range, whereas, for total strain-controlled fatigue tests in the high cycle fatigue (HCF) range, the elastic strain portion ($\varepsilon_{a,e}$) provided through Basquin and the plastic strain portion ($\varepsilon_{a,p}$) from Manson-Coffin are summed up as it is shown in Figure 1.

The failure criterion for those S–N curves, such as fracture, technical crack or reduction in stiffness, has to be defined with respect to the application. The relevant parameters characterising the S–N curve are based on the material, the stress concentration factor, the stress ratio, the load-time function, the environmental condition and the medium. The shape and position of the S–N curve also depend on the microstructure and possibly also on the prior history of the material in the case of ageing.



Figure 1. Description of an S–N curve in the low cycle (LCF) and high cycle fatigue (HCF) regime for total strain-controlled fatigue tests with differentiation into elastic and plastic strain amounts according to Basquin and Manson-Coffin (equations), based on [16].

Depending on the material and manufacturing processes, S–N curves have to be determined with respect to the probability of failure. Especially in the LCF and HCF regime, scatter bands with defined failure probabilities are added into the logarithmic scale by the same slope and the exponent of the mathematical description according to Basquin [17]. Designing a component based on an S–N curve with a failure probability of 50% is critical from a safety point of view if no scatter bands can be provided. Therefore, it is necessary to define a calculated failure probability according to the individual potential risk depending on the application field and the potential of danger for life and system integrity [3,8]. The ASTM E 739 [19] gives useful technical criteria to conduct the statistical analysis of fatigue data, showing, for example, how to calculate confidence intervals for the parameters or for the entire median S–N curve.

Generating fatigue life curves of a material still requires a significant effort and may require performing around 15 to 25 experiments if the curve determined should be considered sufficiently proven, but this is only valid for homogeneous materials and material conditions. In the case of inhomogeneous materials, e.g., casted materials, 40 to 50 and even more specimens need to be tested for statistically verified information regarding the fatigue life [20,21].

In the 1980s, materials data for cyclic loading were collected and compiled in a database in a first approach, which was made available to the engineering designer [20]. In recent years, a couple of different short-time evaluation procedures (STEP) were developed in order to reduce the efforts in experimentation and financial costs for providing fatigue data. Examples for a useful application of STEPs include but are not limited to offshore facilities, bridges or buildings and towers as well as heavy machinery, transportation vehicles, energy generators.

All structural designs today are based on the weakest link model such that the lowest criterion of a design parameter has to be chosen [22]. This criterion is applied for any other component made of the same material although the fatigue lives for each of those different components will vary as a result of the difference in the materials' parameter scatter. It is, therefore, the aim of research to develop methods determining the scattering behaviour of a material for each individual component and which can be applied with justifiable effort and costs. If this is successfully implemented, predictions about the remaining fatigue life of components can be made with considerably higher accuracy. In this study, the new fatigue life calculation method SteBLife (stepped-bar fatigue life) using temperature and magnetic-based measurements as input variables are presented.

2. Materials and Methods

2.1. Materials

Fatigue tests were carried out on AISI/SAE 1045 (1.1191, C45E) and AISI/SAE 4140 (1.7225, 42CrMo4) steel specimens, the materials in the delivered condition had the following characteristics.

• SAE 1045 (C45E) Steel

The chemical composition of the normalised SAE 1045 steel in accordance with DIN EN 10083-1 is shown in Table 1.

Table 1. Chemical composition of SAE 1045 (C45E) steel and comparison with DIN EN 10083-1.

(wt %)		С	Si	Mn	Р	S	Cr	Мо	Ni
DIN standard	Min	0.42	-	0.50	-	-	-	-	-
Din-Standard	Max.	0.50	0.40	0.80	0.030	0.035	0.40	0.100	0.40
Certificate		0.47	0.23	0.72	0.012	0.013	0.67	0.014	0.07

The material was delivered as round bars with a diameter of 20 mm and a scaly surface due to the heat treatment. The heat treatment by the manufacturer consists of austenitising at 850 °C, followed by a controlled slow cooling process to ambient temperature to receive a mostly ferritic-pearlitic microstructure with an average grain size of 16.3 μ m (according to DIN EN ISO 643:2012), which is shown in a light micrograph in Figure 2a.



Figure 2. (a) Light micrograph and (b) transmission electron micrograph of SAE 1045 (C45E) steel in the original state (reproduced from [23], with permission from Elsevier, 2020).

Within transmission electron microscopy investigations in Figure 2b, the dislocation density in the unloaded original state was evaluated to $\rho = 1.2 \times 10^9$ cm⁻², which is a typical value for this material condition [23].

The SAE 1045 steel in the normalisation condition has a Vickers hardness HV10 of 218, a 56 J_m cm⁻² KCU (impact energy for Charpy U-notched specimens) and a mostly ductile behaviour under (quasi)static and cyclic loading. The material's properties are summarised in Table 2, comparing [24–27].

Table 2. Properties	of normalised	SAE 1045	(C45E)	steel.
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R _m (MPa)	R _{p0.2} (MPa)	E (GPa)	A (%)	HV10	KCU ($J_m \text{ cm}^{-2}$)
710	413	204.5	23.5	218	56

SAE 4140 (42CrMo4) Steel

The investigated quenched and tempered SAE 4140 (42CrMo4) steel is in accordance with DIN EN 10083-3 and its chemical composition is given in Table 3.

(wt %)		С	Si	Mn	Р	S	Cr	Мо	Ni
DIN-standard	Min.	0.38	-	0.60	-	0.020	0.90	0.15	-
Diffoundation	Max.	0.45	0.4	0.99	0.025	0.035	1.20	0.30	-
Certificate	e	0.40	0.26	0.74	0.02	0.02	1.07	0.18	0.09

Table 3. Chemical composition of SAE 4140 (42CrMo4) steel and comparison with DIN EN 10083-3.

The material was delivered as round bars with a diameter of 20 mm. The heat treatment by the manufacturer consists of an austenitisation process at 850 °C, followed by quenching in oil and thereafter tempering at 550 °C for 60 min resulting in a characteristic tempered martensitic microstructure with finely dispersed carbides located at the grain boundaries and inside the former austenite grains [28,29]. The microstructure of the low-alloyed steel in the quenched and tempered condition is shown in Figure 3a–c as light, scanning and transmission electron micrograph, respectively). The TEM image in Figure 3c shows the dislocation and carbide structure of the undeformed quenched and tempered material state with rod-shaped or lenticular carbides, which are in shape and size typical for the heat treatment condition of SAE 4140 steel.



Figure 3. (a) Light micrograph, (b) scanning electron micrograph and (c) transmission electron micrograph of quenched and tempered SAE 4140 (42CrMo4) steel in the original state.

From phase analysis by using X-ray (Figure 4), interference lines for Fe-, Cr- and Mo-carbides were observed, which are typical for that material grade in the present heat treatment condition [29].



Figure 4. X-ray phase analysis for the quenched and tempered SAE 4140 (42CrMo4) steel.

Mechanical properties derived from tensile and hardness tests for the SAE 4140 steel investigated can be seen in Table 4. The steel in a quenched and tempered condition shows a Vickers Hardness HV10 of 345 and relatively low ductile behaviour indicated by the values of the elongation at break A and reduction in area Z.

Table 4. Properties of quenched and tempered SAE 4140 (42CrMo4) steel

R _m (MPa)	R _{p0.2} (MPa)	E (GPa)	A (%)	Z (%)	HV10
1051	960	208	16	55	345

2.2. Fatigue Life Calculation Method SteBLife

The SteBLife (stepped-bar fatigue life) approach gives comprehensive information regarding the load-function related deformation behaviour and thus the load-lifetime relation by means of an S-N curve of a material with only a few or even down to one single specimen. The specimen geometry is characterised by a stepped shape with different gauge length diameters (Figure 5) consequently resulting in different local stresses or stress amplitudes along the axial direction of the specimens during loading. The volumes of the individual gauge lengths are the same in order to prevent influences from volume-specific dissipation effects. Moreover, the transitions from one gauge length to another are tangential to reduce notch effects along the specimen to a minimum value. The fatigue tests are performed under constant amplitude loading, and the material response (M) is measured by NDT-related methods. Based on this idea and due to the symmetry of the specimen, it is possible to plot up to five cyclic deformation curves providing the database for cyclic stress material-response (σ_a –M) curves. In contrast to other STEPs such as PhyBaL [30], StressLife [26] or StrainLife [31], no pre-damage from load increase tests (LIT) and strain increase tests (SIT), respectively, due to the previous load steps as well as softening and/or hardening effects from the stepwise increased stress or strain amplitude has to be taken into consideration, whereby the as measured stress material-response relationship can be used, leading to a significant advantage.



Figure 5. Geometry of stepped-bar fatigue life (SteBLife) specimens presented as a (**a**) drawing and (**b**) model.

For the calculation, generalised Morrow [32] and Basquin [17] equations are combined in order to calculate the fatigue life and an S–N dataset for the material investigated.

Following this strategy, three different SteBLife modules were developed up to this date. The first one is SteBLife_{stc} (stc: single test, trend curve) allowing to calculate a trend S–N curve based on one single test with a duration of 2–3 h. The full procedure of the SteBLife_{stc} approach can be seen in Figure 6 [24,27].



Figure 6. Schematic workflow for the fatigue life calculation method SteBLifestc.

SteBLife_{mtc} (mtc: multiple tests, trend curve) is based on 2–5 SteBLife tests at different stress amplitudes in order to improve the statistics of the provided dataset and to generate a representative mean S–N curve (50% failure probability), which also takes the material's inhomogeneity into account.

The third module SteBLife_{msb} (msb: multiple tests, scatter bands) offers the possibility to calculate scatter bands for different failure probabilities, which is an important feature even in the case of more inhomogeneous materials like, e.g., cast irons., therefore, 4–5 SteBLife tests are performed at the same stress amplitude and the obtained lifetimes N_f can be interpreted by a Gaussian distribution allowing to determine the lifetimes for different failure probabilities, as, e.g., 5%, 50% and 95% (or others). The procedures of SteBLife_{mtc} and SteBLife_{msb} are explained more in detail in [25,27,33] as well as based on the results discussed in Section 3.

For the optimisation of the SteBLife specimen geometry, simulations with the COMSOL Multiphysics[®] software were carried out in parallel to the experimental investigations, whereby the dissipated heat of the individual cross-section areas and their heat conduction interaction were considered. This is particularly relevant for the determination of the reference value for the calculation of individual temperature changes ΔT_1 , $\Delta T_{2o/u}$, and $\Delta T_{3o/u}$, which represent the input variables for the subsequent service life calculation according to SteBLife, using temperature measurements as the material response. The thermomechanical modelling comprises the combination of structural mechanics and heat transfer modules that are included in the software library and was carried out for different development stages of the SteBLife specimen geometry.

The first step in the numerical modelling is to develop the mechanical stress fields of the SteBLife specimen for a given cyclic loading. For this purpose, the surface of the specimen geometry was meshed with a variable field network (Figure 7a). Based on this, the mechanical stresses on the surface were simulated for different stress amplitudes (Figure 7b, shown here for $\sigma_a = 300$ MPa in the smallest test cross-section). Since the specimens are cyclically loaded during the fatigue test, a time-dependent consideration was made for a previously defined number of cycles, in which mechanical stresses were modelled for all geometric coordinates in three dimensions as a function of time.



Figure 7. Results of the simulation using COMSOL Multiphysics[®] with the (**a**) model of a SteBLife specimen with mesh network; (**b**) the simulated stress distribution in the specimen for $\sigma_a = 300$ MPa and (**c**) the simulated temperature distribution after a time interval of 10 s during cyclic loading.

The Multiphysics[®] interface in COMSOL calculates the dissipated energy density from the locally present mechanical stresses as a function of time. Using the heat transfer module, temperature fields of the sample surface can be determined (Figure 7c). This approach allows the evaluation of the specimen geometry with respect to the temperature measurement and enables the evaluation/selection of heat conduction models and to determine optimal measuring points for the temperature evaluation. The tetrahedral elements were chosen for these simulations because they are good in the convergence of the numerical solution for curved geometries. The maximum and minimum element sizes were chosen as 1 mm and 0.5 mm respectively. The minimum element quality was chosen as 0.20. The finite element method works by discretising the modelling domains into smaller, simpler, domains called elements. The equations within each element are also known as *shape functions* and can be of different orders. Finally, the solution to these elemental equations approximates the actual solution to the partial differential equation. The user has an option to select the shape functions such as Lagrange and Serendipity to the given problem. In this case, the Quadratic Serendipity function was chosen to minimise the computation time.

2.3. Measurement Technology

2.3.1. Fatigue Testing System

Stress-controlled constant amplitude tests (CAT) using sinusoidal load-time functions at a stress ratio R = -1 and a frequency of f = 5 Hz were carried out at ambient temperature on a servohydraulic testing systems type EHF-U (load range \pm 50 kN) by Shimadzu. For fatigue tests in the HCF regime, the maximum number of cycles N_{max} was limited to 2 × 10⁶.

The material response was measured by means of the change in temperature ΔT measured by an infrared camera and magnetic-based measurands as the Hall voltage ΔU_{Hall} . These quantities are directly related to deformation-induced changes in the microstructure of the bulk material, represent the current fatigue state and provide input variables for the fatigue life calculation. ΔT , ΔU_{Hall} are highly temperature-sensitive, and since metallic materials are very good heat conductors, the microstructure related dissipated energy is superimposed by heat transfer processes requiring temperature stabilised grips (specimen clamping) for the fatigue testing systems. Therefore, a cooling system based on Peltier elements with water coolant was developed. This system is capable of reducing the temperature difference between upper and lower grip during the test to less than 0.5 K as well as the temperature drift during the test to less than 0.5 K.

2.3.2. Temperature Measurements

For the SteBLife approach as described before, the specimen geometry is characterised through a stepped shape with different diameters of the gauge lengths, consequently resulting in different local stress amplitudes σ_a along the rotary axis. For the ΔT evaluation, seven measurement fields were defined within the infrared camera software, one in each gauge length (T₁, T_{20/u}, T_{30/u}, o: upper, u: lower), and two at each shaft (T_{40/u}). ΔT was calculated in accordance with Equations (1)–(3).

$$\Delta T_1 = T_1 - 0.5 \times (T_{40} + T_{4u}) \tag{1}$$

$$\Delta T_{20/u} = T_{20/u} - T_{40/u} \tag{2}$$

$$\Delta T_{30/u} = T_{30/u} - T_{40/u} \tag{3}$$

The diameter of the shafts is larger than the diameter of any other gauge length, so that $T_{4o/u}$ can be considered as the elastic portion of the change in temperature, as well as ambient influences, and can be subtracted from T_1 , $T_{2o/u}$ and $T_{3o/u}$ leading to respective ΔT values, which can be seen as proportional to $\varepsilon_{a,p}$ and are, therefore, related to the evolution of the plastic deformation in each volume of the specimen's gauge lengths.

The theoretical background for the use of temperature measurements in order to characterise the fatigue behaviour can be derived from the consideration of the mechanical stress-strain hysteresis loop. The lines surrounding the hysteresis loop describe the energy that must be applied to plastically deform the specimen during fatigue loading. Since energy cannot simply be extinguished, it is transformed into the internal energy U and the heat energy Q [34]. The energy proportion U enables microstructural changes in the deformed specimen's volume such as dislocation reactions, micro and macrocrack formation as well as their propagation processes. The predominant proportion Q of about 90% from the deformation energy dissipates as heat, which is inter alia related to the released energy from reversed dislocation movement processes and internal friction of new material surfaces due to crack formation and propagation [35]. That is why the temperature change can be correlated directly with the plastic deformation or the cumulative damage. For metallic materials, the change in temperature caused by the energy dissipation can be easily measured on the specimens' surface as a consequence of the high heat conduction rate.

To enable the possibility for a reproducible and a high-precision measurement of ΔT by infrared cameras, the surface of the specimens were painted with high emissivity black paint, obtaining an emission factor of 0.97 and the change in the emissivity was neglected even if the effective temperature changes during the fatigue tests are relatively small (<10 K). For these investigations an infrared camera type thermoIMAGER TIM 450 by Micro-Epsilon was used. This system provides a spectral range of 7.5 to 13 μ m, an optical resolution of 382 × 288 pixels and a thermal sensitivity of 40 mK. The data acquisition software was programmed according to the requirements of the fatigue tests on National Instruments LabVIEW. The software allows to extract thermographs and to measure point/line/field temperatures as numerical matrix information from the specimens' surface under investigation.

For the sake of thermal stability as well as the accuracy during long test intervals, the infrared camera has been modified by an active cooling system.

2.3.3. Magnetic-Based Measurements

Magnetic-based measurements were carried out using Hall probes. According to Edwin Hall, this physical effect is referred to as the Hall effect, which he discovered during his doctoral thesis in 1879 [36].

Since electrons in a solid electrical conductor cannot be deflected completely as a result of the Lorentz force, but only within the conductive material, areas with an excess of electrons and areas with a lack of electrons are generated. This results in a charge separation which, due to the opposite

negative and positive charge surpluses, leads to an electric field with a field strength exerting a force on the electrons which counteracts the Lorentz force F_{Lorentz} .

If both forces are in equilibrium, there is no further charge separation resulting in the Hall voltage U_{Hall} depending on the current I and the external magnetic field B. U_{Hall} indicates magnetic field changes at a constant current and rises linearly with the applied magnetic field and is thereby anti-proportional to the charge carrier density, since a smaller charge carrier density can only lead to an unchanged current intensity at a higher speed of the individual charges. Due to the fact that the charge carrier density in semiconductors is significantly lower than in metals, semiconductors are mainly used for Hall probes. U_{Hall} can be calculated using Equation (4), where A_{Hall} is the Hall coefficient and d is the thickness of the conductor [37].

$$U_{\text{Hall}} = A_{\text{Hall}} \frac{I \cdot B}{d} \tag{4}$$

The usual measuring range of Hall probes is between $0.5 \,\mu$ T and 5 T and depends on the sensor type. For the investigations within the scope of this work, Hall probes type Hal 1822 by Micronas and A1325 by Allegromicro Systems each with a range of -80 to 80 mT, were used. For amplification, a self-developed Hall probe amplifier was applied together with a VirtualBench System by National Instruments.

3. Results and Discussion

3.1. Temperature-Based Calculations

The material response was traced primarily by temperature measurements as described in 2.3.2. As mentioned before, the fatigue tests were performed under constant amplitude loading, whereby a sinusoidal alternating axial force function at a load ratio of R = -1 was used. In the case of the normalised SAE 1045 steel shown in Figure 8a, a force amplitude of $F_a = 8.9$ kN was applied. Based on the symmetry of the specimen's geometry, one cyclic deformation curve for the diameter of 6.0 mm, and each two for the diameters of 6.2 and 6.4 mm are obtained, resulting in local stress amplitudes of 352 (ΔT_{1}), 337 ($\Delta T_{2,o/u}$ o: upper, u: lower) and 318 MPa ($\Delta T_{3,o/u}$). The change in temperature is calculated with respect to the pure elastic loaded shaft of the specimen in accordance with Equations (1)–(3).



Figure 8. (a) Temperature-based cyclic deformation curves for different diameters of the SteBLife specimen and (b) cyclic stress amplitude-temperature curve in a constant amplitude test with $\sigma_a = 352$ MPa (related to $\emptyset = 6.0$ mm) in accordance with the SteBLife_{stc} approach for SAE 1045 (C45E) steel.

Specimen failure occurred during the test at $N_f = 52,320$ (Figure 8a), corresponding to the determined lifetimes with conventional hourglass specimens. The cyclic deformation curves provide the database for cyclic stress material-response curves. For SteBLife_{stc}, ΔT values are extracted at $N_f/2$

and plotted as a cyclic stress amplitude-temperature curve (CST) shown in Figure 8b. In the following, a combination of Morrow and Basquin equations is used for the calculation of the S–N curve (Figure 9) as explained in 2.2 in detail.



Figure 9. Conventional determined S–N data and SteBLife_{stc} curve calculated on the basis of the change in temperature for SAE 1045 (C45E) steel.

Figure 9 gives a first impression regarding the capability of the SteBLife_{stc} approach enabling the calculation of the materials' fatigue life based on one fatigue test only, which is in good accordance compared with the conventionally determined lifetimes from CATs with conventional hourglass specimens. The datapoints of the SteBLife_{stc} curve were calculated for several stress amplitudes by using Basquin's equation as it was shown in Figure 6.

In order to improve the result of the approach and to give it a more statistical meaning, additional SteBLife tests were performed. The overall CST curve of four SteBLife tests is shown in Figure 10a providing the database for SteBLife_{mtc} from which the further calculation is done equivalent to SteBLife_{stc}. Comparing the SteBLife_{mtc} calculation and conventional determined fatigue data, the values for the fatigue strength coefficients are found to be nearly the same (751 MPa for conventional CATs and 743 MPa for SteBLife_{mtc}) and the fatigue strength exponents are -0.07 for both.



Figure 10. (a) Cyclic stress amplitude-temperature curve from constant amplitude tests in accordance with the SteBLife_{mtc} and (b) conventional S–N data and SteBLife_{mtc} curve calculated on the basis of the change in temperature for SAE 1045 (C45E) steel.

The result shown in Figure 10b has the character of an S–N curve for a 50% probability of failure or survival, respectively, and is, therefore, very well suited for materials with a larger scatter behaviour. In the case of more inhomogeneous materials, scatter bands for different failure probabilities

(P_f) may become of interest as additional information in an S–N dataset. Therefore, the SteBLife_{msb}

module was designed and validated, combining 4 to 5 SteBLife tests performed at the same value of σ_a . The probability density ($\varphi(N_f)$) of N_f from performed SteBLife tests can be fitted to a Gaussian distribution, which is in the form of Equation (5). By integrating Equation (5) on an interval from $-\infty$ to N_f , it is then possible to calculate the failure probability P_f (also cumulative distribution, Equation (6)). In this case is μ the mean, represents the N_f value at 50% failure probability; σ^2 the variance, which is the expectation of the squared deviation of a random variable from its mean μ . μ and σ^2 can be determined according to the conducted SteBLife tests as 40,456 and 4.25 \cdot 10^7, respectively. Within the method development, other distribution types as $\arcsin\sqrt{P}$ and Weibull were studied, however, the calculation using Gaussian distribution led to the best results for a general application.

$$\varphi_{(N_f)} = \frac{1}{\sqrt{2\pi\sigma^2}} \cdot \exp\left(\frac{-\left(N_f - \mu\right)^2}{2\sigma^2}\right)$$
(5)

$$P_{f} = \frac{1}{2} \left(1 + erf\left(\frac{N_{f} - \mu}{\sqrt{2\sigma^{2}}}\right) \right)$$
(6)

From the Gaussian distribution, the lifetimes of different failure probabilities can be calculated (Figure 11a). In this example, 5%, 50% and 95% were chosen for plotting the scatter bands, but others can be calculated as well, e.g., 0.01% since this is an important value for the design of components such as in the automotive industry. Different fatigue strength exponents can be received from the five SteBLife tests and can be plotted as an exponent-lifetime relation. By using the calculated lifetimes for the failure probabilities of interest in an exponent-lifetime relation, fatigue strength exponents for a failure probability of 5%, 50% and 95% can be derived (Figure 11b).



Figure 11. (**a**) Gaussian distribution and (**b**) fatigue strength exponent-lifetime relation for five tests in accordance with the SteBLife_{msb} approach for SAE 1045 (C45E) steel.

Based on the results of Figure 11, S–N curves for failure probabilities of 5, 50 and 95% can be calculated, which are shown in Figure 12 in comparison with S–N data from 11 CATs performed with conventional hourglass specimens used as reference values. Figure 12 underlines that the conventionally determined lifetimes can be described very reliably by scatter bands evaluated on the basis of SteBLife_{msb}. With a view to the mathematical description according to Basquin, the fatigue strength coefficients and exponents of calculated and conventionally determined S–N data are in the same range.



Figure 12. Conventionally determined S–N data and SteBLife_{msb} curve with calculated scatter bands for different failure probabilities evaluated from five SteBLife tests for SAE 1045 (C45E) steel.

Within current research, the SteBLife_{msb} procedure has also been applied to other materials and material conditions such as normalised as well as quenched and tempered SAE 1015 (C15E) steel, quenched and tempered SAE 1045 (C45E) steel and normalised as well as quenched and tempered SAE 4140 (42CrMo4) steel in order to validate the general applicability of the method for this group of materials.

These research activities are still in progress and not yet fully completed, but Figure 13 gives a further outlook on more results of SteBLife calculations. Figure 13 shows a comparison of conventionally determined fatigue data with the S–N curves calculated by SteBLife_{msb} with related scatter bands for failure probabilities of 5%, 50% and 95% for quenched and tempered SAE 4140 steel.



Figure 13. Conventionally determined S–N data and SteBLife_{msb} curve with calculated scatter bands for different failure probabilities from five SteBLife tests for SAE 4140 (42CrMo4).

Due to the material's higher strength condition, the material responses are rather small, which means that their detection requires very stable boundary conditions and very high-resolution measurement techniques. During the fatigue tests, the ΔT slopes only show changes of a maximum of 0.5 K, which was challenging, but managed by the active cooling of the clamps and a further improvement of the infrared camera technique.

It is noteworthy that despite the very small temperature changes, the calculation using SteBLife_{msb} led to an excellent result. Considering that the 5 tests carried out for this purpose replace a number of tests of 30 to 40, the enormous potential with regard to the reduction of experimental time and costs can be deduced from the result obtained.

In addition to temperature measurements using infrared cameras, selected measurements were also carried out with a self-developed Hall sensor array. For magnetisation, a direct current of 3 A was applied to the specimen, which resulted in microstructure-dependent magnetic fields along the different gauge length diameters during the fatigue tests, which were recorded with the Hall sensor array. The sensor head and the positioning of the Hall elements were adapted to the SteBLife specimen geometry in such a way that identical distances to the specimen surface can be realised for the three cross-sectional diameters during measurement. By using a high-precision measuring card in combination with a LabVIEW data acquisition software, the change in the magnetic field was measured in terms of voltage values. The complete setup is shown in Figure 14a.



Figure 14. (a) Test setup with Hall sensor array, (b) cyclic deformation curves based on the tangential magnetic field for a SteBLife test with $F_a = 9.85$ kN and (c) fatigue life calculation according to the SteBLife_{stc} method based on the magnetic field measurement for the normalised SAE 1045 (C45E) steel.

Figure 14b shows the changes in the tangential magnetic field during a SteBLife test for a specimen of the normalised SAE 1045 steel. The test was carried out with a constant force amplitude of $F_a = 9.85$ kN, resulting in a stress amplitude of $\sigma_a = 373$ MPa in the smallest cross-section (5.8 mm) and values of 337 MPa and 306 MPa in the cross-sections with 6.1 and 6.4 mm. In accordance with the results derived from temperature measurements, the cyclic deformation curves show locally different behaviour with varying incubation intervals as well as maximum values assigned to the respective local stress amplitude. Figure 14c shows the result of the fatigue life calculation according to SteBLife_{stc} based on the change of the tangential magnetic field. As with the calculation based on temperature measurement, a very good agreement between the conventionally determined fatigue data and the calculation result was achieved. This result underlines the flexibility of the SteBLife fatigue life calculation method, which can be applied in a variety of ways, including the use of other measurement methods.

4. Conclusions

This paper shows how measurement techniques based on temperature and magnetic field can be included in the short-term evaluation procedure SteBLife (stepped-bar fatigue life) for the advanced evaluation of fatigue data in an appropriate way. The reduction in effort, time and cost achieved with such an approach can help the present material condition within an existing engineering structure to be evaluated, such as it is becoming increasingly necessary with ageing structures. Within the scope of this paper, the SteBLife approach is presented on the example of (i) normalised SAE 1045 (C45E) and

(ii) quenched and tempered SAE 4140 (42CrMo4) steel in order to provide fatigue data in a fast and efficient manner. Therefore, the calculation is based on input parameters derived from temperature measurements through infrared cameras as well as through self-developed Hall probe arrays.

Through the application of the SteBLife approach, the duration and cost for determining a complete S–N dataset on an experimental basis is reduced by more than 90% when compared to state-of-the-art approaches. This results from the fact that principally only a low number of experiments in the high cycle fatigue range are required, which can be performed within a few hours each.

The advantage of combining the new experiment and calculation approach with the above-mentioned measurement techniques is that damage is not related to mechanical stress and strain only, but has rather been extended to a variety of other physical parameters, would those be of a thermal, electric, electromagnetic or acoustic nature. Those parameters, which may react much more sensitive to submicroscopic damage such as dislocation reactions, are a source to improve the understanding of a nonlinear damage accumulation process.

Furthermore, material properties determined by these techniques are principally independent of a material volume considered as long as the scale of this volume exceeds the size in which the inhomogeneous portions within a microstructure have an influence. As such it may, therefore, be applicable to complex geometries like notched specimens or even components as well as structures. The physical quantities monitored can be transferred from those localised and potentially critical spots within the structures to a reference database and can be directly related to microstructural changes in a bulk material of which the fatigue loading properties have been obtained with an approach such as SteBLife.

Author Contributions: Conceptualisation, P.S., H.W., R.A. and F.W.; methodology, P.S. and H.W.; software, H.W. and R.A.; validation, P.S., H.W., R.A. and J.T.; formal analysis, H.W., R.A. and R.S.V.; investigation, H.W., R.A. and F.We.; resources, P.S.; writing—original draft preparation, P.S., R.A.; writing—review and editing, R.A., F.W., J.T. and H.W.; visualisation, H.W., R.A. and R.S.V.; supervision, P.S.; project administration, H.W.; funding acquisition, P.S. All authors have read and agreed to the published version of the manuscript.

Funding: The authors gratefully acknowledge the financial support for this study from the German Research Foundation (Deutsche Forschungsgemeinschaft, DFG) under project STA 1133/6-1.

Acknowledgments: The authors would like to thank Shimadzu Europe and Micro-Epsilon for their support in technical equipment provision.

Conflicts of Interest: The authors declare no conflict of interest.

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